



INERT WALL THERMAL FEEDBACK VALIDATION STUDY

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Supported by the Ministry of Interior, Safety research programme of the Czech Republic
2015 - 2020 (BV III/1 - VS) under number VI20192022120
Fire development modelling of engineered wood.



Motivation – Fire spread modeling

- Engineered **W**ood **P**roducts (MDF, OSB, plywood, particleboard)
- CFD coupled to pyrolysis model
 - Thermal decomposition properties
 - Material properties
 - Thermo-physical properties
 - Optical properties
 - Model dependent parameters

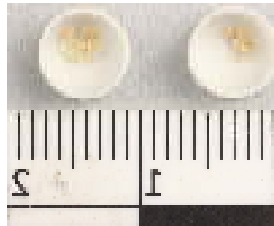




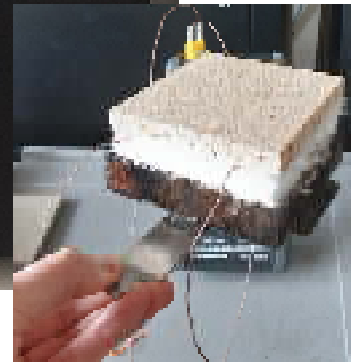
Pyrolysis model input parameters



➤ Micro-scale



➤ Bench-scale



Calibration
versus
validation?

How to validate pyrolysis model of EWP?



Vertical flame spread experiment

- 2 × 1 meter board
- 50 × 5 × 10 cm propane burner

Wong, William Chiu-Kit. 2012. CFD Flame Spread Model Validation: Multi-Component Data Set Framework. : Worcester Polytechnic Institute

Combustible wall fire model

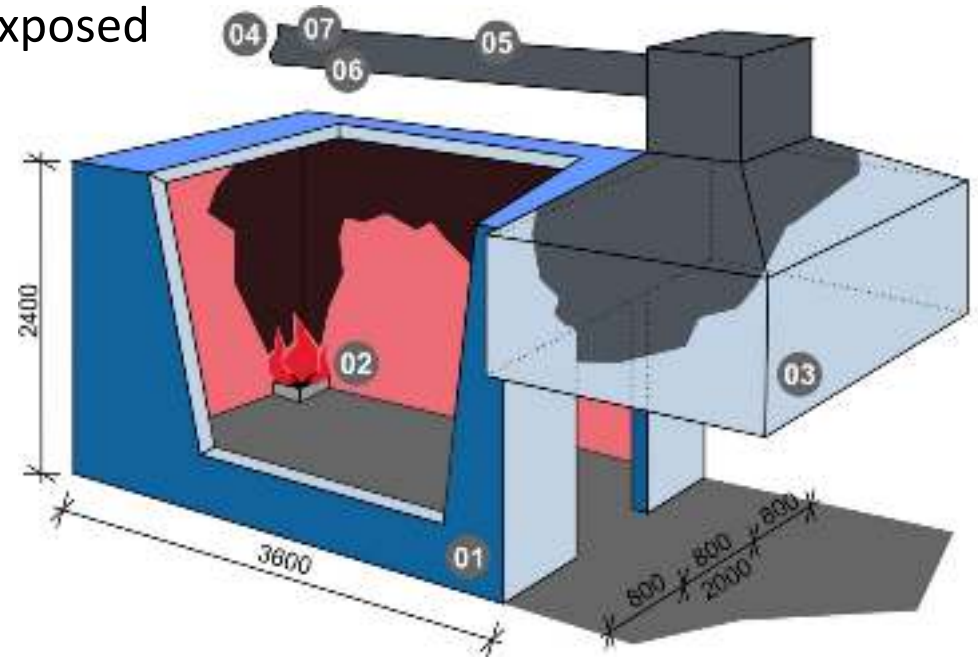
- Turbulent buoyant fluid flow
- Gas phase kinetics
- Flame heat transfer
- Pyrolysis

Inert wall fire – thermal feedback to the wall



Fiber cement panel

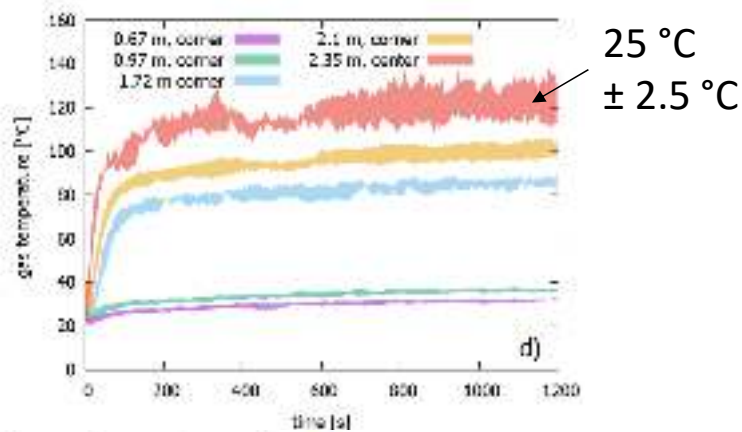
- 2 × 1 meter
- 12 mm thick
- Back side exposed



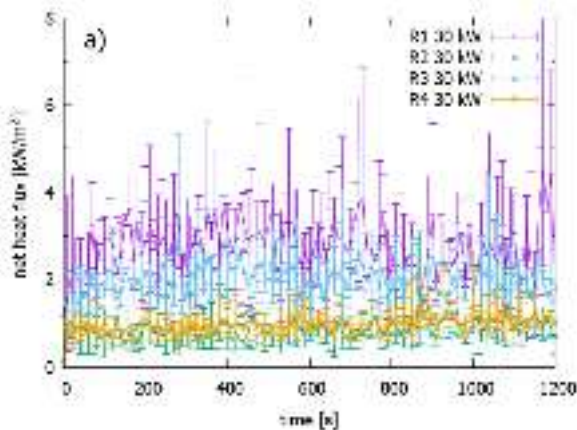
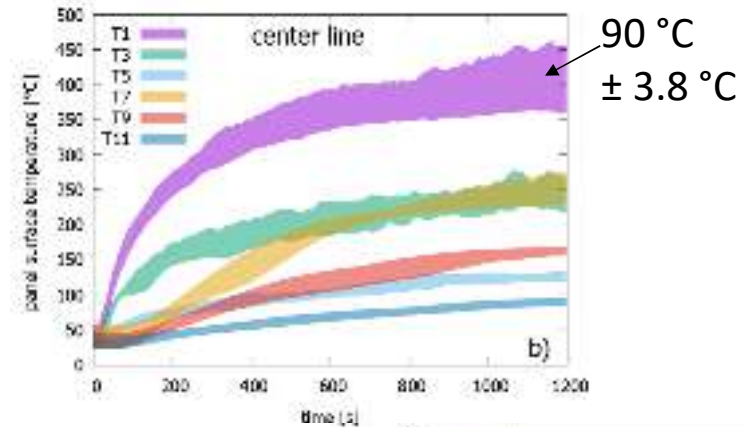


Repeatability and measurement uncertainty

Temperature in the room



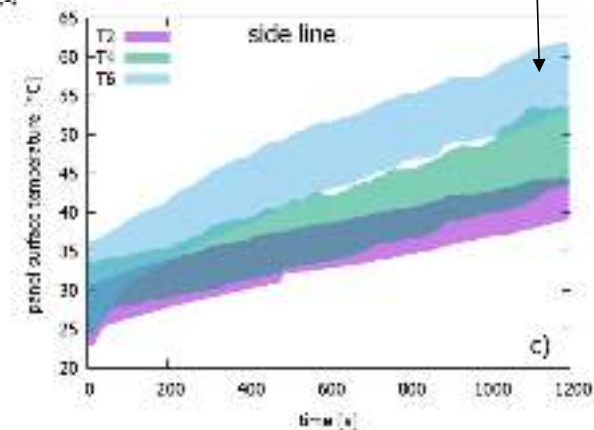
Panel surface temperature



Net heat flux

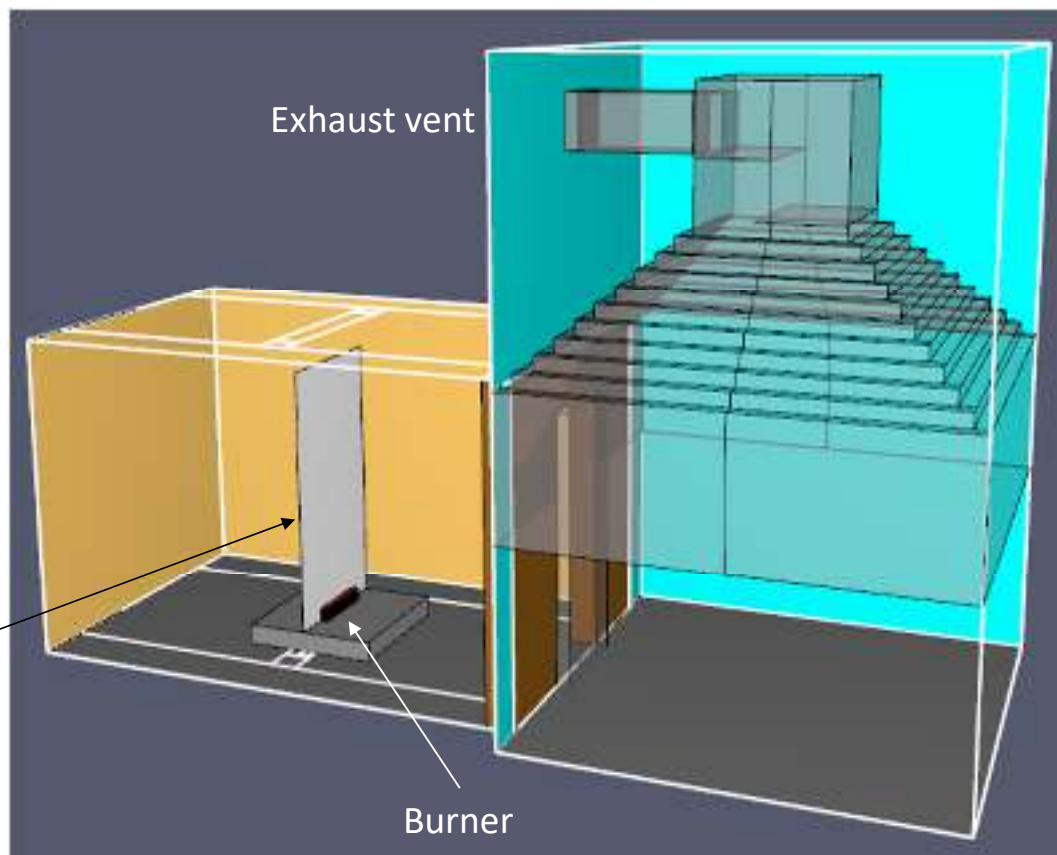
- Uncertainty of the heat flux gauge?
 - Calibration accuracy is $\pm 3\%$ at full scale
 - Non-linearity error 10 to 50 %
- Net heat flux not symmetrical along the center line

10 °C
± 2.5 °C





FDS model



- 1 layer, 1 cell thick
- k, ρ, c_p constant
- Backing exposed



Grid resolution – 5 cases

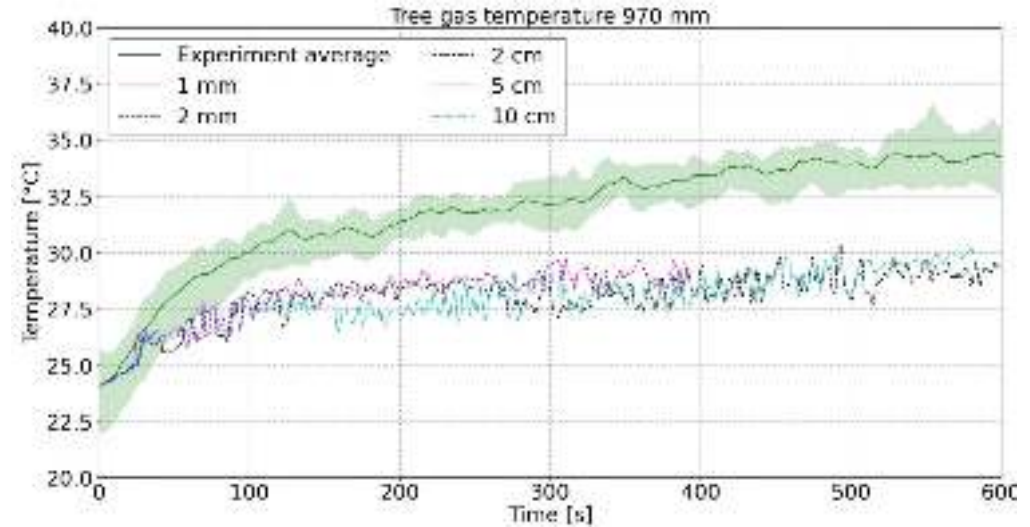
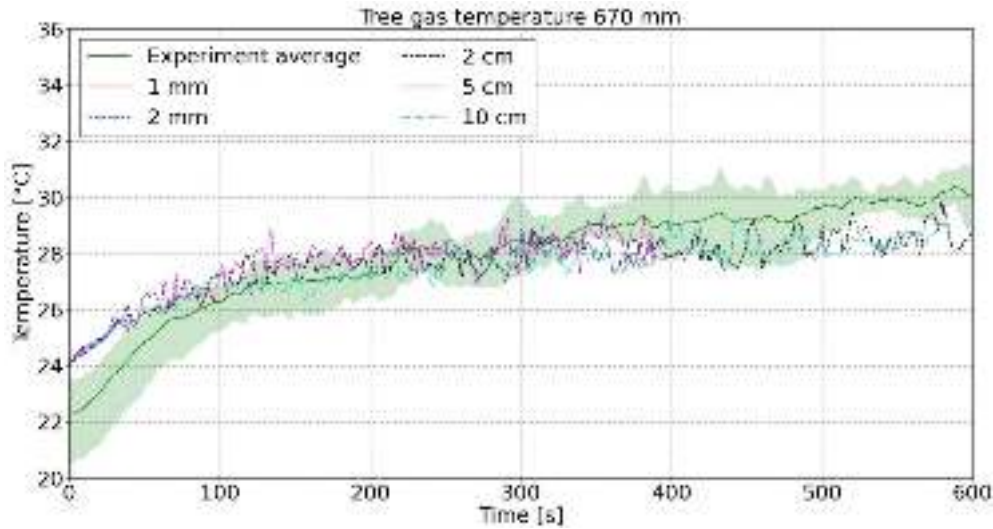
Brown, A., Bruns, M., Gollner, M., Hewson, J., Maragkos, G., Marshall, A., McDermott, R., Merci, B., Rogaume, T., Stoliarov, S., Torero, J., Trouvé, A., Wang, Y., Weckman, E. (2018). Proceedings of the First Workshop Organized by the IAFSS Working Group on Measurement and Computation of Fire Phenomena (MaCFP), *Fire Safety Journal*, 101, 1-17.

- Grid resolution required to resolve boundary layer and wall gradients is in an order of 1 mm
- High resolution grid reduce numerical errors - the discrepancies between experimental data and computational results may be attributed to modeling errors

Mesh ID	Grid cell size in direction perpendicular to the panel (number of cells in this direction)	Total number of cells	Number of meshes	Simulation time (s)
1 mm	1mm (12) - 2mm (26) - 4mm (9) - 2cm (95)	65 902 500	80	10
2 mm	2 mm (20) - 4 mm (10) - 8 mm (10) - 4cm (46)	22 222 800	36	40
2 cm	1cm (20) - 2cm (10) - 10 cm (17)	804 084	11	600
5 cm	1 cm (20) - 5 cm (38)	1 070 496	6	394
10 cm	1 cm (20) - 10 cm (19)	637 812	6	585



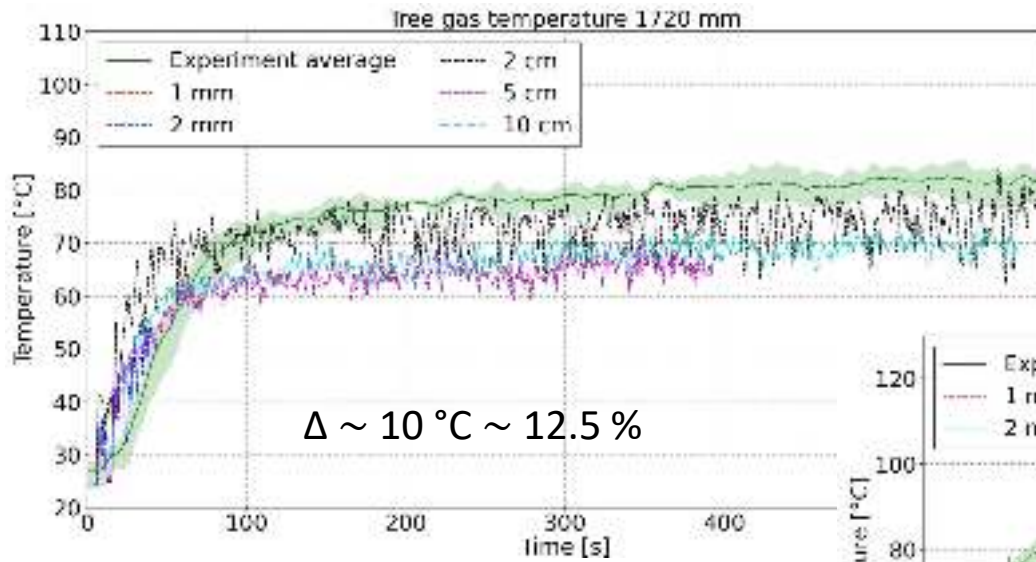
Cold layer temperature



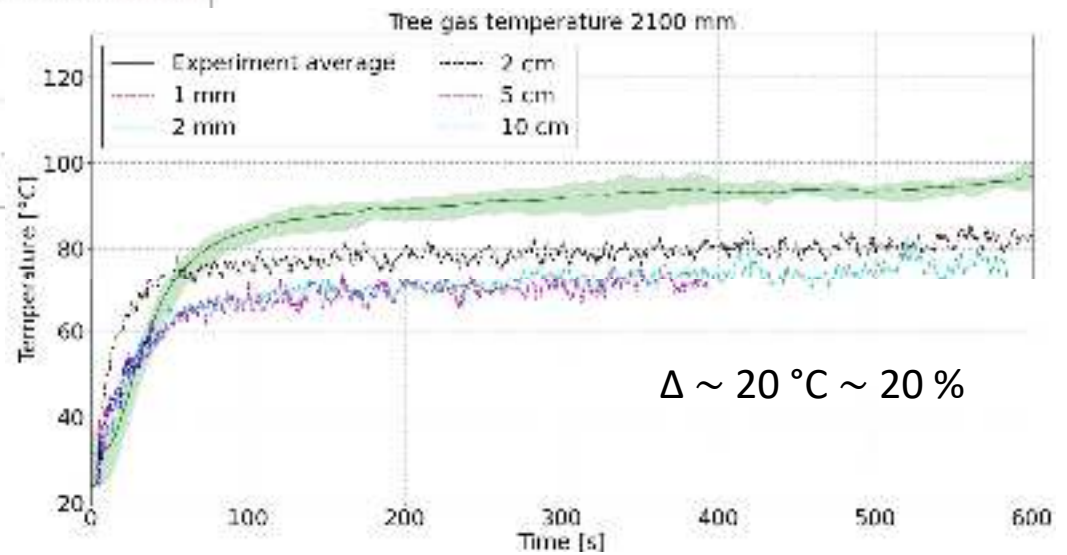
$\Delta \sim 5 \text{ }^\circ\text{C} \sim 15 \%$



Hot layer temperature

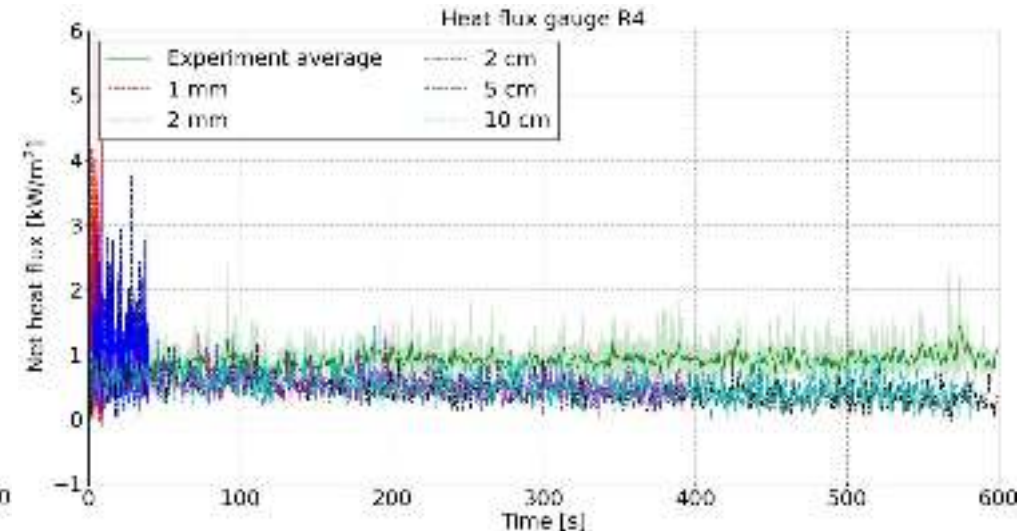
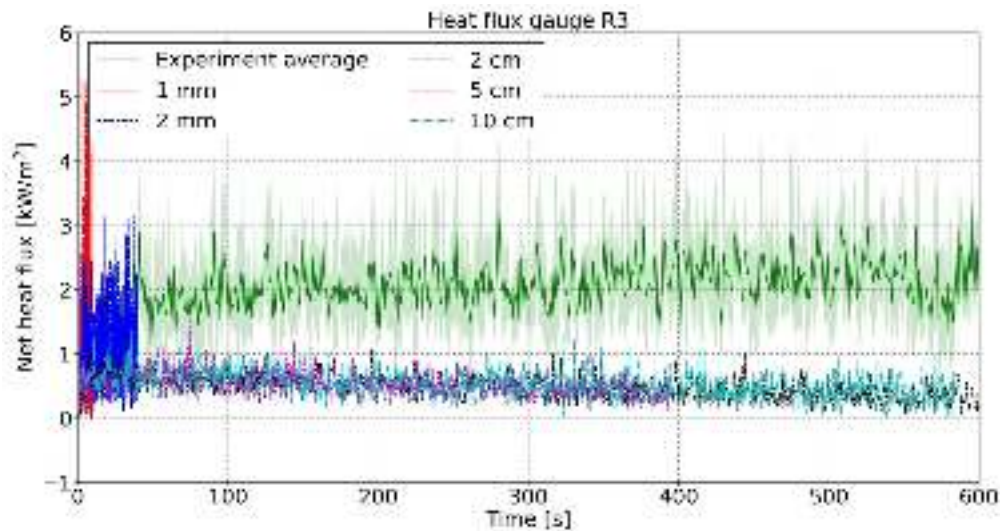


$\Delta \sim 10 \text{ }^\circ\text{C} \sim 12.5 \%$

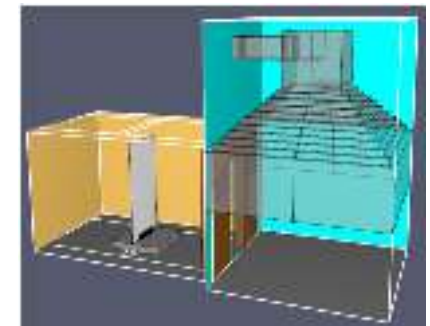


$\Delta \sim 20 \text{ }^\circ\text{C} \sim 20 \%$

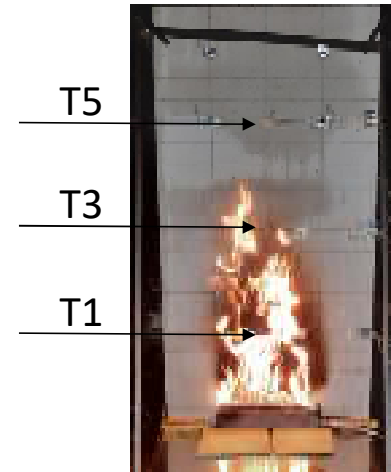
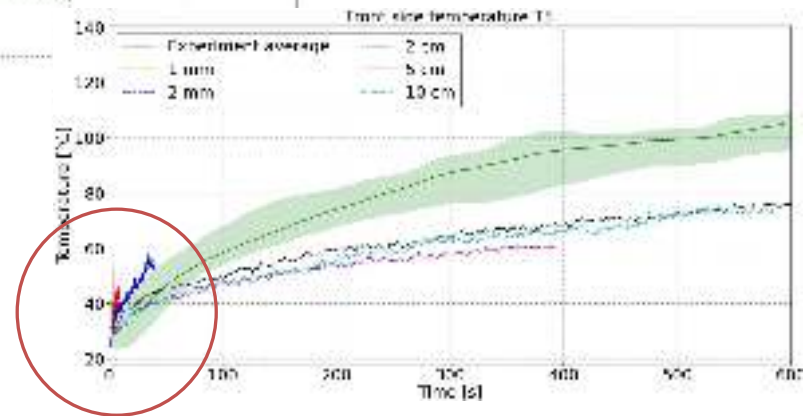
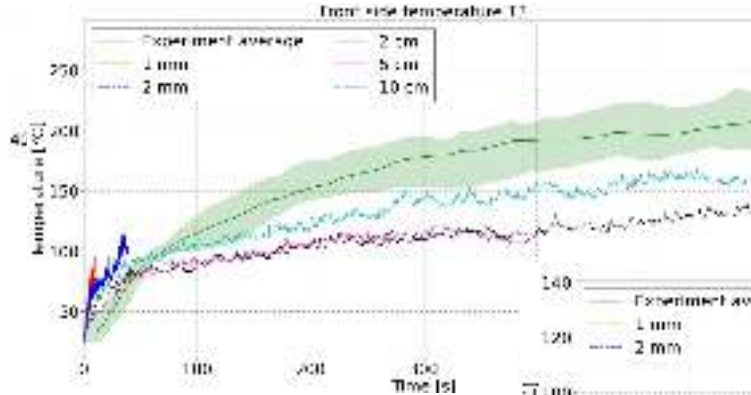
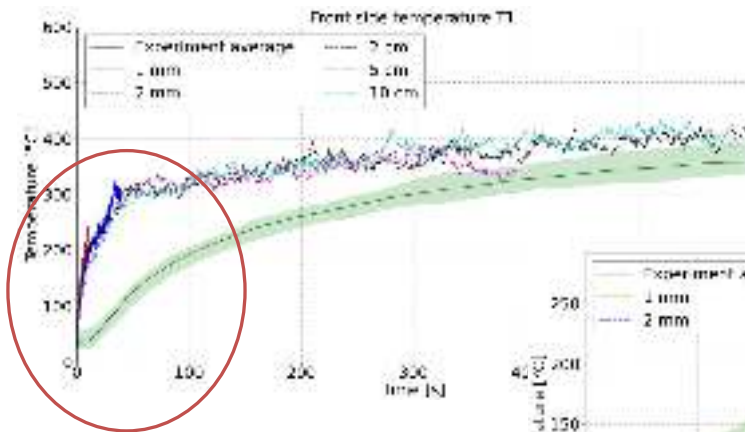
Net heat flux to the wall



- Heat flux in the model is symmetrical along the panel center line
- Wrong boundary condition in the hood area
- Net heat flux to the cold versus hot wall

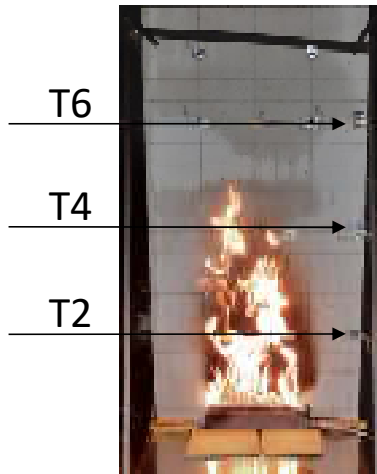


Panel surface temperature – center line

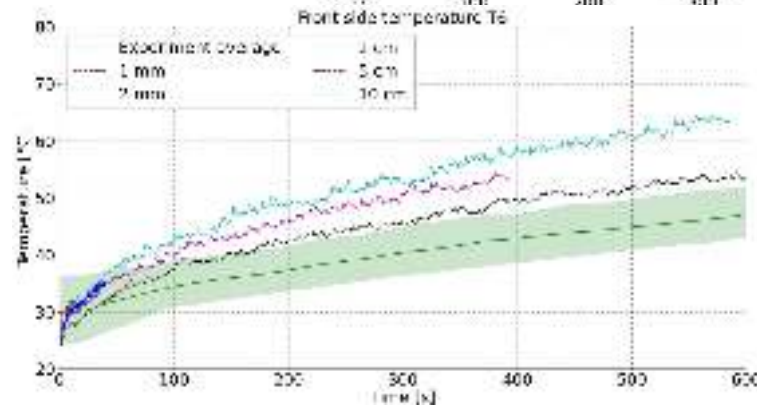
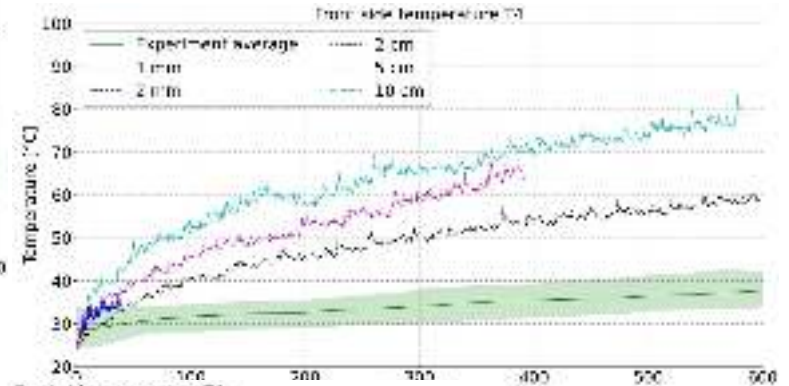
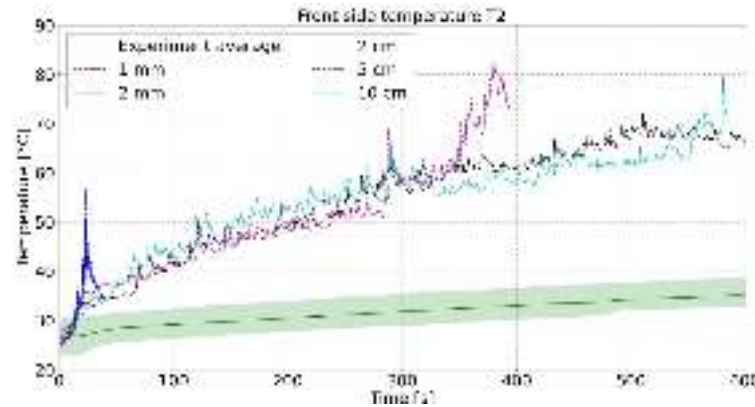


- Grid resolution
- Presence of moisture
- k, c_p are temperature dependent
- 1D heat transfer

Panel surface temperature - side line



- Unsymmetrical flow conditions
- Presence of moisture
- k, c_p are temperature dependent
- 1D heat transfer





Conclusions

Experiments

- Evaluate the HFG uncertainty
- Measure velocity profile in the doors to the room

Model

- Change boundary condition in the hood area
- Add temperature dependent k , c_p
- Evaluate the „conductive losses“ upwards and to the side of the panel - 3D heat transfer.
- Get at least 200 seconds of the very fine grid resolution simulation